

Experimental results on air permeability of agricultural nets

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Abstract

In order to evaluate the influence of the texture characteristics of agricultural nets on the air flow passing through them, a micro wind tunnel was designed and built in the testing and engineering laboratory of Sachim s.r.l., an Italian high-density polyethylene (HDPE) technical textiles manufacturer. The micro wind tunnel (0.1345 m diameter) allowed the simultaneous measurement of the air flowrate and the pressure drop through a net sample. The equipment was designed to set up the inclination of the net samples with respect of the airflow at fixed angles (90°, 60°, 45°, 30°). The variation of the pressure drop with the air velocity on four different flat woven round monofilament HDPE nets perpendicular to the air flow are here reported. In all cases the air velocity measured into the micro wind tunnel was above 4 m s⁻¹ hence reported results do not consider low wind speed.

Preliminarily, the loss coefficient was assumed as a function of porosity and Reynolds number, calculated with reference to the equivalent diameter of the pores, $F_s(Re, \epsilon)$ and showed percent deviations

form experimental results in the range 14.6%÷25.3%. Hence a simplified expression of the loss coefficient, depending only on the porosity was proposed, $F_s(\epsilon)$ which highlighted differences with experimental results in the range 3.5%÷20.3%. Finally, results were compared with those based on the Bernoulli's principle found in the literature.

Introduction

Plastic nets are widely used in various agricultural applications: protection from hail, wind, snow or strong rainfall in fruit-farming and ornamentals, shading nets for greenhouses and moderate influence of the microenvironment around a crop are the most common cases (Castellano *et al.*, 2008). In some cases nets are placed on the vents of the structure, in others they cover the entire structure, the latter being this kind of structure called either net-house or screen-house. Usually, such net-covered structures do not have ventilation openings since it is expected that the cover is sufficiently permeable to provide enough ventilation capacity whilst protecting the crop against wind, hail or sun radiation.

The air permeability characteristics of the covering material determine both the ventilation performance - together with buoyancy and convective phenomena (Teitel, 2007) - of a nethouse as well as its windbreak effect together with aerodynamic effects induced by the geometry of the structure (Robertson *et al.*, 2002; Mistriotis and Castellano, 2012).

When nets are placed on the vents of the greenhouse, they affect mainly the internal microclimate due to the fact that usually vents are closed in case of strong wind. In any case, it is strategic to understand the correlation between the airflow passing through the net and its geometrical characteristics in order to properly design the building both from a structural and an energy efficiency point of view.

In the literature, the air flow characteristics through the net have been evaluated either in terms of permeability based on the motion equation of a fluid through a porous material expressed by the Forchheimer equation (Miguel *et al.*, 1997, 1998, 2001; Bartzanas *et al.*, 2002; Fatnassi *et al.*, 2003; Valera *et al.*, 2005), or in terms of coefficient of discharge based on Bernoulli's flow theory (Brundrett, 1993; Kosmos *et al.*, 1993; Pearson and Owen, 1994; Kittas *et al.*, 2002; Bailey *et al.*, 2003; Fatnassi *et al.*, 2003; Ishizuka *et al.*, 2005; Wang *et al.*, 2007).

In many cases, the pressure drop given by both methods differs little, considering that the difference in the results obtained from applying Forchheimer or Bernoulli equations is less significant than the uncertainty of not knowing the air pressure distribution around the greenhouse, a parameter which is needed to calculate the ventilation rate through an opening.

In order to evaluate the influence of construction parameters of

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agricultural nets on the air flow, a micro wind tunnel was built - based, when compliant with the nets characteristics, on UNI EN ISO 9237 recommendations - in the Testing and Engineering Laboratory at Sachim s.r.l., an Italian high-density polyethylene (HDPE) technical textiles manufacturer. Experimental results in terms of loss coefficient of four flat woven - a simple orthogonal weave between weft and warp threads (Castellano *et al.*, 2008) - round monofilament HDPE nets are reported in this paper.

Models for fluid flow

Teitel (2007) presented a comprehensive literature review on studies concerning the airflow resistance characteristics of various nets. The following section is mainly based on his review.

Forchheimer equation

The steady state, incompressible fluid flow through a highly porous material in which the volume of the solid matrix exceeds that of the fluid contained within it can be expressed by the Forchheimer equation:

$$\frac{\mu}{K} u + \rho \left(\frac{Y}{K^{0.5}} \right) |u|u = \frac{dp}{dx} \quad (1)$$

The permeability K represents the ability of the medium to transmit the fluid through it. Eq. (1) is derived from the general motion equation of one-dimensional mass transfer through a permeable material (Miguel *et al.*, 1997). Eq. (1) shows how fluid velocity is related to pressure drop, through the viscous resistance force, which appears due to momentum transfer at the fluid interface (μ/K) and the pore inertia effects ($\rho Y/K^{0.5}$). The viscous resistance is predominant for low velocities of the fluid through the porous material ($\mu_r = \mu/\varepsilon$), when the spaces occupied by the fluid are smaller than those occupied by the solid matrix and when the length of the porous material is comparable with its cross section (Bejan, 2013).

Some authors, considering a net equivalent to a porous media, used Eq. (1) to describe the airflow thorough a net. To analyse the airflow characteristics of greenhouse screening materials, and to determine their permeability K and inertial factor Y , Miguel *et al.* (1997) and Miguel (1998) tested several screens in a wind tunnel, and their findings let Miguel (1998) conclude that the correlations that best relate the screen permeability and inertial factor to the porosity ε were:

$$K = 3.44 \cdot 10^{-9} \varepsilon^{1.6} \quad (2)$$

$$Y = 4.30 \cdot 10^{-2} \varepsilon^{-2.13} \quad (3)$$

Different correlation equations were given by Valera *et al.* (2005) who tested eleven screens:

$$K = 5.68 \cdot 10^{-8} \varepsilon^{3.68} \quad (4)$$

$$Y = 5.67 \cdot 10^{-2} \varepsilon^{-1.1604} \quad (5)$$

The porosity ε is a geometric property defined as the ratio of non-solid volume (voids) to the total volume of the net and can be measured using image processing (Castellano *et al.*, 2008).

The motion is described by the Reynold number Re , which can be interpreted as the ratio between inertial and viscous forces:

$$Re = \frac{\rho u d}{\mu} \quad (6)$$

In the general motion equation of one-dimensional mass transfer through a permeable material d is assumed coincident with the diameter of the particles of the solid matrix (Bejan, 2013). In formulation concerning the airflow passing through a net, d represents either the mesh size (distance between wires, pore equivalent diameter, *etc.*) or the wires diameter.

For very small velocities ($Re < 1$) the quadratic term of Eq. (1) can be neglected obtaining the Darcy law, which defines the permeability:

$$\frac{dp}{dx} = \frac{\mu}{K} u \quad (7)$$

Increasing the airflow velocity - Miguel (1997) assumed empirically as threshold value $Re < 150$ - convective inertia effects become predominant, the linear term of the Eq. (1) can be neglected and the pressure drop can be described only by the quadratic term of Eq. (1) leading to Bernoulli's formulation:

$$\Delta p = 0.5 \frac{\rho}{C_d^2} u^2 \quad (8)$$

where the characteristics of the porous media are enclosed into the discharge coefficient C_d .

The porous media approach has been widely used in recent years in simulating nets in computational fluid dynamics analyses of greenhouse microclimate, the source term in the momentum equation is generally represented by Eq. (1) (Teitel, 2007).

Coefficient of discharge

The discharge coefficient approach - based on Bernoulli's theory - is derived from the so-called orifice analogy. In the orifice model, the open area between the filaments is treated as a submerged orifice and, depending on the geometry of the opening, a discharge coefficient is determined for the pore (Pedersen, 1969).

Lu *et al.* (1996) developed an orifice model for calculating the permeability of monofilament fabrics in which the discharge coefficient was presented as follows:

$$C_d = \sqrt{\frac{\rho u^2 (1 - \alpha_c^2)}{2 \Delta p \alpha_c^2}} \quad (9)$$

Effective fractional open area (EFOA) is given by $a_c = A_c / P_r$. Since the infancy of the orifice model, discharge coefficient has been related to the flow Reynolds number in the following form (Pedersen, 1969; Gooijer *et al.*, 2003; Wakeman and Tarleton, 2005):

$$C_d = k_c Re^{m_c} \quad (10)$$

Note that Reynolds number is defined differently in different studies. Following the work of Pedersen (1969), Lu *et al.* (1996) defined their Reynolds number based on the wetted perimeter of the orifice W_c through which the air passes:

$$Re_w = \frac{4 \rho u l_f^2}{W_c \mu} \quad (11)$$

The EFOA and W_c were expressed by:

$$\alpha_c = \frac{\sqrt{\varphi_f^2+1}}{2\varphi_f} + \frac{\varphi_f}{2} \ln \left(\frac{1+\sqrt{\varphi_f^2+1}}{\varphi_f} \right) - \frac{1}{\varphi_f} - \frac{\sqrt{\varphi_f^2+1}}{\varphi_f^2} \left(1 - \frac{1}{\varphi_f} \right) \quad (12)$$

$$W_c = 2d_f \sqrt{\varphi_f^2 + 1} \left[\left(1 - \frac{1}{\varphi_f} \right) E \left(k, \frac{\pi}{2} \right) + 1 \right] \quad (13)$$

In Eq. (13), $E(\kappa, \pi/2)$ is the elliptical integral of the second kind where:

$$\varphi_f = \frac{l_f}{d_f} \quad (14)$$

$$k = 1 - \frac{\varphi_f^2}{(1+\varphi_f^2)(\varphi_f-1)^2} \quad (15)$$

A simplified form of the discharge coefficient was used by Rigakis *et al.* (2015) in order to estimate the ventilation rate in screenhouses:

$$\Delta p = 0.5 \frac{\rho u^2}{C_d^2} = 0.5 \frac{\rho u_i^2}{\varepsilon^2 C_d^2} \quad (16)$$

Bailey *et al.* (2003), in order to evaluate the airflow resistance of greenhouse ventilators with insect screens, used the relation proposed by Brundrett (1993):

$$\Delta p = F_s \frac{1}{2} \rho \left(\frac{Q}{A_0} \right)^2 \quad (17)$$

where the loss coefficient F_s , is directly correlated to the discharge coefficient (commonly used to quantify the flow resistance of an opening) with the following:

$$C_d = 1/\sqrt{F_s} \quad (18)$$

and the loss coefficient was expressed by

$$F_s = g(Re)h(\varepsilon) \quad (19)$$

with

$$g(Re) = \frac{w}{Re} + \frac{q}{\log(Re+1.25)} + r \log(Re) \quad (20)$$

and $h(\varepsilon)$ defining the influence of the screen porosity ε , expressed as

$$h(\varepsilon) = \frac{(1-\varepsilon^2)}{\varepsilon^2} \quad (21)$$

Brundrett (1993) showed that Eq. (21) fitted the data better than did other alternative expressions such as $1 - \varepsilon^2$ and $(1 - \varepsilon)^2$ that have

been cited in other literature (Richards and Robinson, 1999). In this case, the Reynolds numbers were based on the diameter of the wires forming the screen. The first term in Eq. (20) dominates when $Re < 1$; the third term provides the nearly constant value at high Reynolds numbers $Re > 200$; the second term provides the transition between the first and third terms.

Bailey *et al.* (2003) used the Eq. (19) as the basis for correlating pressure drop coefficients of five nets and suggested, based on their experimental results, different coefficient with respect to Brundrett (1993) (Table 1).

Pinker and Herbert (1967), according to Eq. (19), suggested to split F_s into two independent components as well: a screen porosity function, $h(\varepsilon)$, and a Reynolds number function, $g(Re)$. With reference to $g(Re)$, they tested four functions and concluded that it was difficult to discriminate between the four since the particular form of $h(\varepsilon)$ was more important than that of $g(Re)$.

Ishizuka *et al.* (2005) used a vertical, open-ended tube with the test screen across the top, and an open, spiral, electrical heating wire across the bottom. The heat input and temperature difference across the test screen under steady-state conditions were measured and the thermally created pressure difference was calculated. Measurements were made on nine wire mesh screens with porosities of 0.371, 0.504 and 0.778 over the Reynolds number range (based on the wire diameter) 0.4 to 95, and the loss coefficients were correlated by:

$$F_s = 40 \left[Re \frac{\varepsilon^2}{1-\varepsilon} \right]^{-0.65} \quad (22)$$

Lopez Martinez *et al.* (2014) derived the coefficient F_s equalling Eqs. (1) and (17) and assuming $dp/dx = \Delta p/e$, (Molina-Aiz *et al.*, 2009):

$$F_s = \frac{2e}{k^{0.5}} \left(\frac{1}{Re_p} + Y \right) \quad (23)$$

This coefficient was used to predict the pressure drop through the mesh for Reynolds numbers $Re_p < 10^5$ (Teitel, 2001). Re_p is the Reynolds number based on the screen permeability and can be calculated by considering the square root of specific permeability K (Lopez Martinez *et al.*, 2014):

$$Re_p = \frac{\sqrt{K} u \rho}{\mu} \quad (24)$$

Coefficients K and Y depend mainly on the porosity of the net, according, for example, to Eqs. (2), (3), (4) and (5).

When the approaching flow is not perpendicular to the screen, Laws and Livesey (1978) proposed that the loss coefficient $F_{s,\beta}$, for a flow with an approach angle of β could be expressed as:

$$F_{s,\beta} = F_s \cos^2 \beta \quad (25)$$

Table 1. Values of coefficients w , q , r proposed by Brundrett (1993) and Baily (2003) in the equation of the function $g(Re)$.

	w	q	r
Brundrett (1993)	7.125	0.88	0.055
Bailey (2003)	18	0.75	0.055

where F_s is the loss coefficient for flow perpendicular to the screen ($\beta=0^\circ$), the expression is valid for flows within $\beta=45^\circ$.

Eq. (25) can be considered a simplified relation with respect to that proposed by Pinker and Herbert (1967) and improved by Brundrett (1993), in which the loss coefficient depends on the approach angle as well:

$$F_{s,\beta} = F_s \cos^2 \beta \quad (26)$$

which implies that for a given porosity and Reynolds number, the value of the pressure-loss coefficient is expected to decrease when the flow is not perpendicular to the screen (Teitel, 2007).

Teitel and Shklyar (1998) showed that β is not the only parameter affecting the pressure drop. They emphasised the importance of whole geometry by indicating that, when the distance between two adjacent threads of a woven screen is smaller than five times the thread diameter, the texture of the screen affects both the pressure drop through it and the downstream flow pattern.

To many practical purposes, it is quite complicate to evaluate experimentally empirical coefficients in Eq. (1), and differences among the various studies were larger for the values of K and Y than for those of F_s (Teitel, 2007). The choice of either the Forchheimer or Bernoulli equations makes little difference and, as the pore dimension of the screen was increased, the difference between the values calculated by the two equations tended to decrease. Kittas *et al.* (2002) used both the porous medium method and Bernoulli's equation to calculate the pressure-loss coefficient of a screen with a porosity of 0.6, which they tested, and the values obtained by the respective methods differed by only about 3%. Teitel (2001) compared the two methods and concluded that they agreed quite well in their predictions of the pressure drop through screens. For this reason, in this study a discharge coefficient approach was adopted. Based on experimental results, the loss coefficient of tested net was calculated as a function of the Reynolds number and the porosity of the net, $F_s(Re, \epsilon)$. In addition, a simplified expression of the loss coefficient, depending only on the porosity was proposed, $F_s(\epsilon)$. Calculated values of the loss coefficient were compared with those proposed by different authors found in the literature and with experimental results.

Materials and methods

Laboratory devices

A micro wind tunnel was purposely designed and built at Sachim s.r.l.'s testing and engineering laboratory. The iron steel wind tunnel with a circular section ($D=0.1345$ m) allowed varying the airflow velocity in the range $0 \div 15$ m/s (Figure 1). The pressure drop upstream and downstream the fabric specimen was measured by means of a manometer mod. Aerofiltri MM200600 able to appreciate a pressure difference in the interval of $0 \div 200 \pm 5$ Pa. In addition, the test setup allowed modifying the inclination of the plastic net sample J inside the wind tunnel (90° when perpendicular, 60° , 45° , 30°) with respect of the airflow (Castellano *et al.*, 2015).

The air flowrate through the wind tunnel was measured with a hot wire airflow sensor mod. SMC PF2A 712H in series with the net to be characterised. Distributed pressure drop due to the roughness of the inner surface of the pipe was taken into account. It was calculated by means of three measurements without the net sample at different air velocity, average values allowed to evaluate the relationship between the distributed pressure drop and the air velocity and, finally, the distributed pressure drop was subtracted to experimental data gathered with net samples.

The velocity - to be used for fluid dynamics calculations - was then calculated with respect to the tunnel cross section. The ambient temperature was in the range of $20 \pm 1^\circ\text{C}$, compliant with the sensor specifications.

Tested nets

A set of flat woven round monofilament HDPE nets with different geometrical characteristics was tested (Table 2). Four nets with the same thickness of the warp and weft thread and different porosity were part of set A1 ($d_{warp}=0.28$ mm) (Table 1). Some of tested nets were commercial models, other were built specifically for the tests.

Each net was tested three times and the average value was taken into account in calculations. In the present paper, results concerning the airflow perpendicular to the net sample are reported.



Figure 1. Rendering of the device used to measure the air permeability of nets and fabrics at different inclinations. Specimen holder to set up the different inclinations: the picture refers to the specimen holder 60° .

Results and discussion

For all the investigated nets, a second order correlation between the air velocity measured in the wind tunnel and the pressure drop Δp was found (Figure 2).

A parabolic curve formally equal to Eq. (17) was assumed, the coefficients which better fit measured data were evaluated by means of the ordinary least squares method:

$$\Delta p = \frac{\rho}{2} F_s u^2 \quad (27)$$

where $\rho=1.205 \text{ kg m}^{-3}$ at 20°C . A very high coefficient of determination $R^2>0.99$ was gathered for all tested nets (Figure 2). Eq. (27) represents a parabola with the vertex coincident with the origin of axes; the shape of the curve depends on the value of the coefficient F_s . The lower is F_s , the wider is the concavity of the parabola throughout the whole velocity range measured in the micro-wind tunnel.

The loss coefficient was investigated firstly when the flow was perpendicular to the sample. Preliminarily, according to Eq. (19), F_s was assumed as the product of two different functions: $h(\epsilon)$ depending on the porosity and $g(Re)$ depending on the Reynolds number. Concerning $h(\epsilon)$, was confirmed the expression proposed by Brundrett (1993), a very high coefficient of determination ($R^2>0.87$) was found assuming:

$$h(\epsilon) = \left(\frac{1-\epsilon^2}{\epsilon^2}\right) \quad (28)$$

A distribution based on empirical coefficients proposed by Brundrett (1993) (Table 1) was assumed as $g(Re)$ but, differently from Brundrett (1993), Bailey *et al.* (2003), in Eq. (20) the Reynolds numbers were calculated with reference to the equivalent diameter of the pores:

$$Re_l = \frac{\rho u l_{eq}}{\mu} \quad (29)$$

where: $l_{eq}=4l_{warp}l_{weft}/(2l_{warp}+2l_{weft})$ is the equivalent diameter of the pores (Table 2).

Finally, the loss coefficient of tested nets was expressed as:

$$F_s(Re_l, \epsilon) = \left[\frac{7.125}{Re_l} + \frac{0.88}{\log(Re_l+1.25)} + 0.055 \log(Re_l) \right] \left(\frac{1-\epsilon^2}{\epsilon^2}\right) \quad (30)$$

Due to the high determination factor ($R^2>0.87$) highlighted by $h(\epsilon)$ with respect to F_s , a simplified expression of the loss coefficient, depending only on the porosity, was proposed as alternative to Eq. (30):

$$F_s(\epsilon) = 0.74 \left(\frac{1-\epsilon^2}{\epsilon^2}\right) \quad (31)$$

The empirical coefficient 0.74 in Eq. (31) was calculated by means of the least square method, comparing experimental data with calculated applying Eq. (27) and assuming the loss coefficient depending only by the porosity of the net.

In Figure 3, pressure drops across the tested nets, induced by different values of the superficial velocity, evaluated according to Eq. (20) proposed by Bailey *et al.* (2003), Eq. (22) proposed by Brundrett (1993), Eq. (23) proposed by Ishizuka (2005), were plotted. In the same diagrams, the measured values of the pressure drop in the micro wind tunnel and the calculated values according to Eqs. (30) and (31) were reported.

Formulation proposed by Bailey *et al.* (2003) and Brundrett (1993) provided results very similar to each other (Figure 3) and a good correlation with experimental results (Table 3). Percentage errors lower than 13.0 and 12.4% respectively were observed in both formulations in samples A1-N1, A1-N2 and A3-N3 (Table 3). Higher differences were observed, 20.5 and 25.5% respectively, in measurements of net sample A1-N4 (Table 3). The same behaviour, with higher percentage errors between measured and calculated values applying Eq. (30), was observed. Such result depended on the function $g(Re)$ proposed by Brundrett or Bailey which decreases hyperbolically from infinity to almost 0.5 when the Reynolds number passes from 0 to 100 and is almost constant when $Re>100$, consequently in the range of investigated nets - $76<Re<282$ - high variability of $g(Re)$ is gathered, especially in nets with low porosity characterised by lowest Reynolds numbers. The formulation proposed by Ishizuka *et al.* (2005) overestimated the

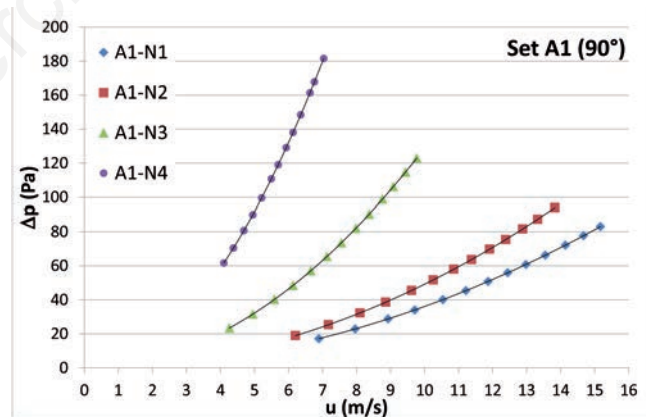


Figure 2. Experimental values (marked ones) of velocity u , and pressure drop Δp for the tested nets. The solid lines indicate the parabolic regression curves

Table 2. The set of the nets under investigations. d_{warp} is the thickness of the warp; d_{weft} is the dweft thickness of the weft; l_{warp} is the distance between the threads of the warps; l_{weft} is the distance between the threads of the weft; l_{eq} is the equivalent diameter of the pores, units are in millimetres, ϵ is the porosity of the net.

Set	Net Id.	d_{warp}	d_{weft}	l_{warp}	l_{weft}	l_{eq}	ϵ
A1	N1	0.28	0.28	1.39	1.72	1.54	71.6%
A1	N2	0.28	0.28	0.97	1.54	1.19	65.7%
A1	N3	0.28	0.28	0.35	1.54	0.57	47.0%
A1	N4	0.28	0.28	0.27	0.49	0.35	34.4%

pressure drop through the net with respect to experimental results (Figure 3). Calculated values with Eq. (22) fitted better the experimental results at high air velocity as confirmed by high values of standard deviations. The best fitting was observed comparing calculated values with those measured at net sample A1-N4 with a percentage error of 7.5% (Table 3).

Eq. (30), which $F_S(Re, \epsilon)$ was assumed for, highlighted percentage errors with experimental data less than 25.3%, higher respect to those calculated by formulation proposed by Bailey *et al.* (2003) and Brundrett (1993) (Table 3).

The simplified formulation, described by Eq. (31), which assumed F_S depending only on the porosity, showed a good matching with measured data, similar to those obtained applying Eq. (30) except for net sample A1-N4 for which the best fitting with a percentage error 3.5% was observed (Table 3).

Conclusions

The airflow through the nets was experimentally evaluated using a purposely-built micro wind tunnel able to modify the inclination of the sample with respect to the airflow rate. In the present paper, results concerning the airflow perpendicular to the net sample are reported.

The airflow motion was described using the Bernoulli equation in terms of loss coefficient. Results confirmed those available in the literature in terms of: i) dependence of the pressure drop on the square of the velocity and the porosity by means of the parameter $(1-\epsilon^2/\epsilon^2)$; ii) the negligibility of the of the loss coefficient dependence on Reynolds for ordinary values of the air velocity.

Preliminarily, in this study the loss coefficient was assumed as the

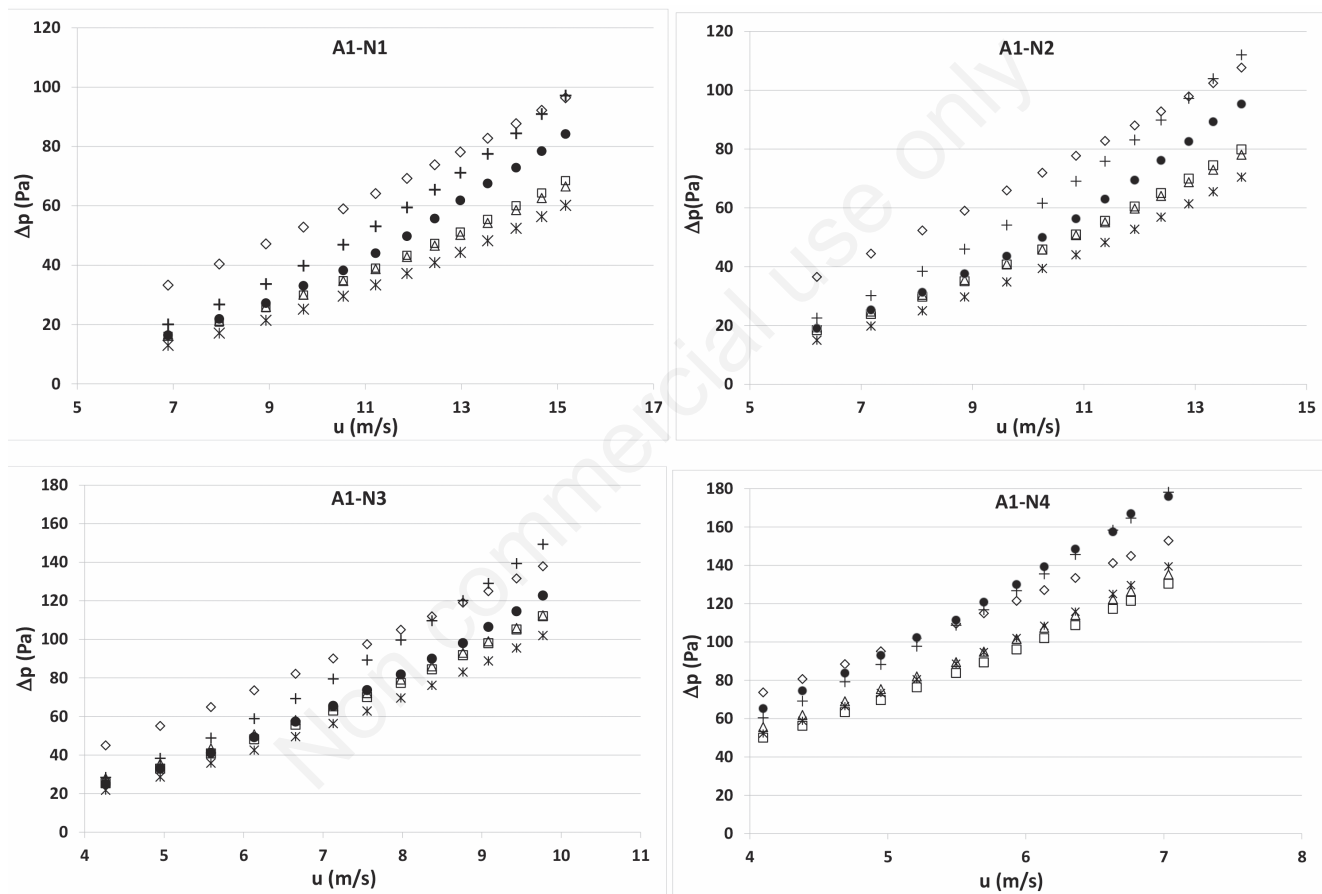


Figure 3. Pressure drop Δp across tested nets induced by different velocity of the airflow u . (●) Measured in the wind tunnel; (◇) Bailey *et al.* (2003); (□) Brundrett (1993); (△) Ishizuka *et al.* (2005); (×) Equation (30) assuming $F_S(Re, \epsilon)$; (+) Equation (31) assuming $F_S(\epsilon)$.

Table 3. Differences between calculated and experimental values normalised with respect to the experimental values and associated standard deviations.

Sample	Bailey <i>et al.</i> (2003)	Brundrett (1993)	Ishizuka <i>et al.</i> (2005)	$F_S(Re, \epsilon)$	$F_S(\epsilon)$
A1-N1	13.0±6.7	12.4±5.4	45.6±26.7	25.3±2.8	18.9±3.2
A1-N2	10.2±5.9	10.2±4.5	42.5±24.0	22.9±2.3	20.3±2.5
A1-N3	5.3±3.3	4.6±2.7	37.5±20.5	14.6±1.4	20.3±2.3
A1-N4	20.5±2.7	25.5±1.1	7.5±4.3	21.2±0.7	3.5±2.2

product of two different functions: $h(\varepsilon)$ depending on the porosity and $g(Re)$ depending on the Reynolds number, $F_s(Re, \varepsilon)$, and showed percent deviations from experimental results in the range 14.6%–25.3%. Differently from previous papers, the Reynolds number was calculated with reference to the equivalent diameter of the pores and not to the diameter of the wire. Hence, due to the high determination factor ($R^2 > 0.87$) highlighted by $h(\varepsilon)$ with respect to F_s , a simplified expression of the loss coefficient, depending only on the porosity was proposed, $F_s(\varepsilon)$. The simplified formulation of the loss coefficient, allowed to predict the pressure drop with respect to the air velocity with a good accuracy and highlighted differences with experimental results in the range 3.5%–20.3%. The two formulations of the loss coefficient, $F_s(Re, \varepsilon)$ and $F_s(\varepsilon)$, were compared with those based on the Bernoulli's principle found in the literature such as Bailey *et al.* (2003), Brundrett (1993) and Ishizuka *et al.* (2005). Results showed that the relationship proposed by Brundrett (1993) and Bailey *et al.* (2003) were in very good agreement with experimental results. Highest differences, 25.5 and 20.5% respectively, were observed in the net sample characterised by the lower porosity. The equation, proposed by and Ishizuka *et al.* (2005), overestimated the pressure drop at higher porosity and the fitting with experimental results increased when the porosity decreased, differences were in the interval 7.5–45.6%. The discharge coefficient approach allowed correlating the airflow characteristic uniquely to geometric characteristics of the net such as the porosity and the equivalent diameter of the pores (by means of the Reynolds number). As a consequence, evaluating the pressure drop with respect to the air speed simply knowing the geometric characteristic has been proved as possible even if further are needed to deepen the effect of other parameters such as the geometry of the orifice, the diameter of threads or the kind of the weave. In any case, at this moment it seems more effective to evaluate the airflow characteristics of the net by means of experimental results into wind tunnel. Further study on other sets of textiles will be performed in the next future in order to validate the results here presented and to investigate the correlation of the loss coefficient with respect to the other geometrical parameters of the net.

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Nomenclature

Variables and parameters

A_C	area of the pores, mm ²
C_d	discharge coefficient
D	diameter of the micro wind tunnel, m
d_f	filaments diameter, mm
d_{warp}	diameter of the warp, mm
d_{weft}	diameter of the weft, mm
e	thickness of the mesh, mm
F_S	loss coefficient
F_{st}	loss coefficient as function of the approach angle of the flow
$h(\varepsilon)$	function of the porosity
$g(Re)$	function of the Reynolds number
K	permeability parameter of a net, m ²
k_c	coefficient in Equation (10)
k, q_f	coefficients in evaluation of EFOA
l_{eq}	equivalent diameter of the pores, mm
l_f	filaments center-to-center distance, mm
l_{warp}	distance between the threads of the warp, mm
l_{weft}	distance between the threads of the weft, mm
p	pressure, Pa
Q	volumetric flow rate, m ³ s ⁻¹
R	coefficient of correlation
R^2	coefficient of determination
Re	Reynolds number
Re_d	Reynolds number based on the diameter of the particles of the solid matrix
Re_l	Reynolds number based on the equivalent diameter of the pores
Re_p	Reynolds number based on the screen permeability
u	fluid velocity, m/s
u_i	fluid velocity through the net, m/s
w, q, r	empirical coefficients
W_c	wetted perimeter of the orifice, mm
x	direction of the mono dimensional flow motion
Y	inertial factor

Greek letters

α_c, α_C	EFOA
β	approach angle of the flow with respect to the screen (0° if perpendicular), deg
Δp	pressure drop, Pa
ε	porosity
μ	dynamic viscosity, kg m ⁻¹ s ⁻¹
θ	inclination of sample into the micro wind tunnel with respect to the airflow (90° when perpendicular), deg
ρ	density, kg m ⁻³